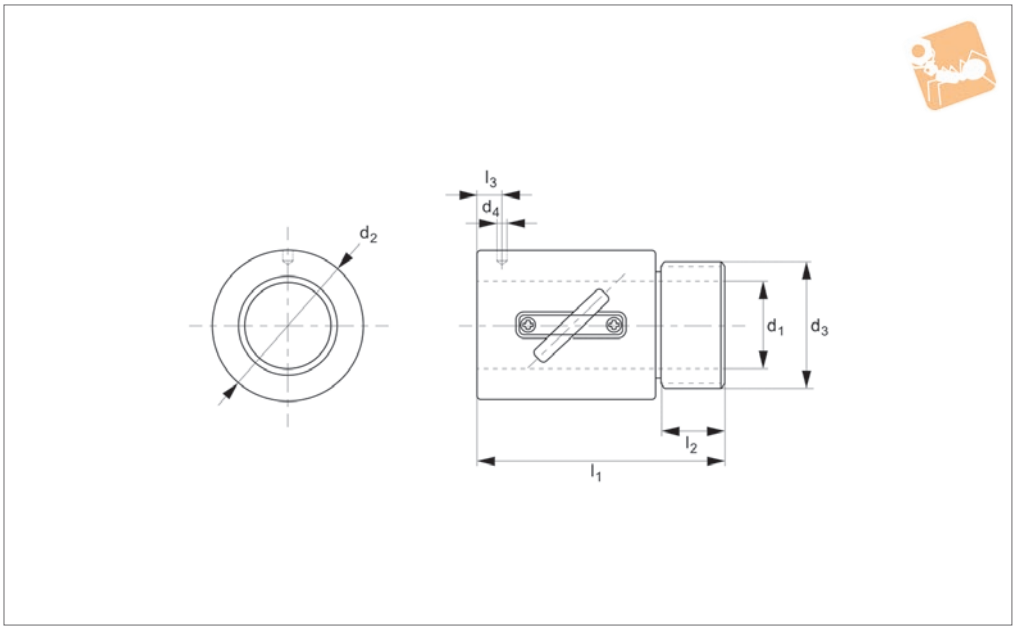


L1379.C

Material
Steel body (16MnCr5), balls (100Cr6) and polyurethane (Vulkolan) seals.

Technical Notes
Axial clearance 0.05mm.
Preload max. 5% of dynamic load.

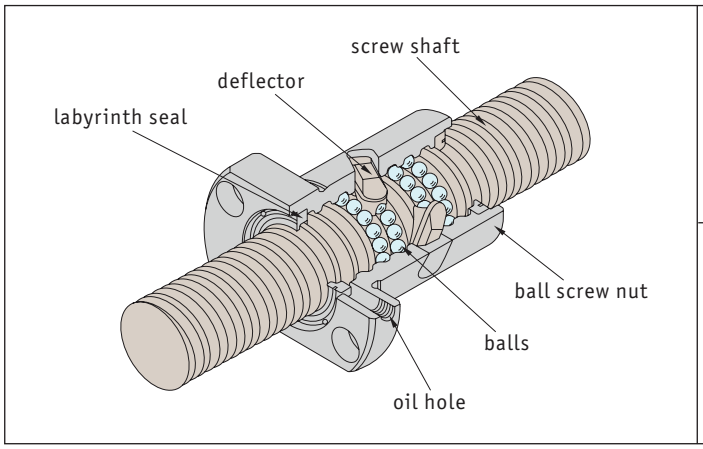
For axial run-out, concentricity and parallelism figures see technical pages.
For use with miniature ball screws L1379.

Tips
Fit ball nut to screw using the sleeve provided. Offer up the ball nut to the screw and slide carefully on. Do not remove the ball nut from the sleeve provided - the ball bearings can come loose rendering the ball nut unusable.

Order No.	d_1 for screw	Pitch	d_2 tol. G6	d_3	d_4	l_1 ±0.15	l_2	l_3	No. of circuits	Ball dia.	Dyn. load C kN max.	Static load C_0 kN max.	Stiffness N/µm
L1379.C08-250	8	2.5	17.5	M 15x1P	3.0	23.5	7.5	10.0	2.5x1	1.2	1.85	3.73	167
L1379.C10-020	10	2.0	19.5	M 17x1P	3.2	22.0	7.5	3.0	3.5x1	1.2	2.72	6.51	167
L1379.C10-040	10	4.0	25.0	M 20x1P	3.0	34.0	10.0	3.0	2.5x1	2.0	3.92	7.39	137
L1379.C12-040	12	4.0	25.5	M 20x1P	3.0	34.0	10.0	13.0	3.5x1	2.5	7.88	16.16	226
L1379.C12-050	12	5.0	25.5	M 20x1P	3.0	39.0	10.0	16.3	3.5x1	2.5	7.85	16.11	235
L1379.C14-040	14	4.0	32.1	M 25x1.5P	3.0	35.0	10.0	11.0	3.0x1	2.5	7.33	15.77	235

Ball Screws

Overview



L1370

L1388

L1375

L1390

Standard ball screws

L1370

L1371

L1372

L1373

Miniature ball screws

L1379.F

L1379.C

Rolled ball screws

	Pitch (travel per revolution)						
Ø	5	10	16	20	25	40	50
16	●	●	●				
20	●	●		●			
25	●	●			●		
32	●	●		●			
40	●	●		●		●	
50		●		●			●
63		●		●			
80		●		●			

Miniature ball screws

	Pitch (travel per revolution)					
Ø	1	2	2.5	4	5	Nut
6	●					flanged
8	●	●	●			flanged
10		●		●		flanged/cylinder
12		●		●	●	flanged/cylinder
14		●				flanged/cylinder

When selecting a ball screw some of the main factors to consider are:

- Maximum required travel speed
- Maximum axial compression (buckling load)
- Method of support of the ball screws
- Type of unit required, flanged, cylindrical etc.

In general it is best to support the ball screws with our ball screw support units (L1388 to L1406) with a fixed end (generally where the motor is mounted) and a floating (support) end. The support units are selected to suit the loads likely to be required, the size of the ball screw (especially its core diameter) and the type of mounting required. Details of the machining required for each end of the ball screw are shown in the bearing mounts technical section.

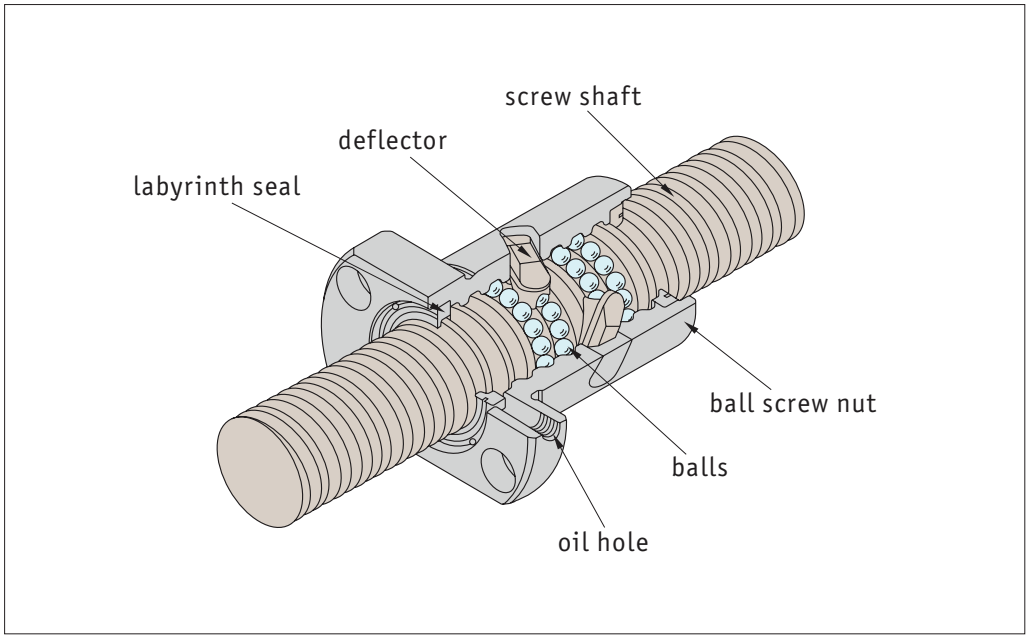
The data table for the ball screws show the diameter, the lead of the ball screw (i.e. how far the nut travels for one complete revolution of the screw) as well as the mass moment of inertia (also known as the rotational moment of inertia) - this is the extent to which an object resists rotational acceleration about its axis.

Maximum speeds and buckling load data are shown in the technical pages.

When using a ball screw the ambient temperature should not exceed +80°C.

During assembly, the parallel alignment of the guides should be ensure.

The details on the concentricity of the ball nuts to the ball screws are shown on the technical pages. For linear guideways for use with ball screws please see our part numbers L1016 etc.



Lubrication - the ball screws must be adequately lubricated. This is dependent on load, speed, motion sequence and temperature. Do not use lubricants containing Mo/So or graphite.

Ball Screws from Automotion Components

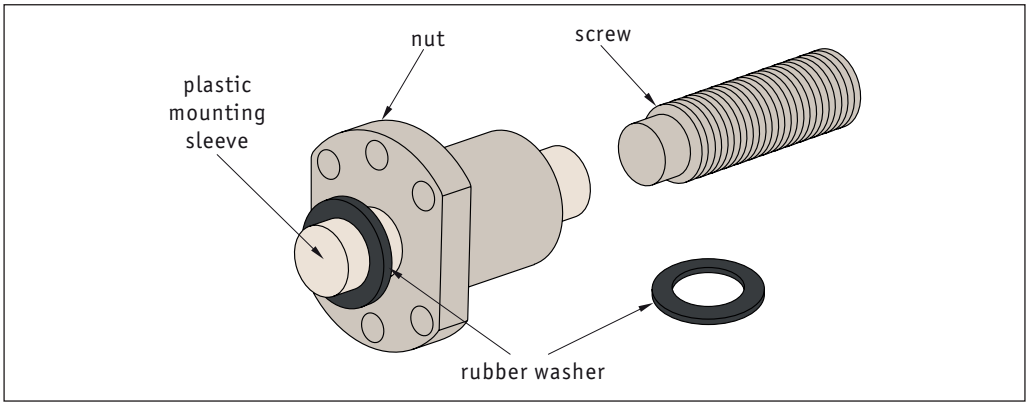
Ball Screws

Care instructions

In general, the ball nut is already on the ball screw and should not be removed. If you need to machine the ball screw, then the plastic mounting sleeve should be used to retain the ball bearings whilst the nut is removed.

Mounting the nut on the screw

Sometimes ball screws are delivered with a separate ball nut. When mounting the nut on to the screw take care as if done incorrectly the ball bearings may come off the ball nut.



Ball nuts should be mounted only with the help of a plastic mounting sleeve (delivered with the nut). The start of the thread should be aligned so that the seal and the internal parts of the nut are not damaged.

1. Remove the rubber washer from one side of the sleeve. Push on the nut with the sleeve on the end of the screw. Press the sleeve against the start of the screw thread.
2. Screw the nut onto the thread using a slight axial pressure, then screw the nut on for it's entire length.
3. Remove the mounting sleeve only when the nut is completely threaded on to the screw.
4. Lock the nut on to the screw (to prevent any unscrewing) using an O ring or similar - whilst installing the system.

If the balls do unfortunately escape...

1. Pick them up (the nut is only compatible with the original balls). The load capacity can still be achieved if one or two balls are missing.
2. Carefully clean all parts, use the sleeve as a mounting jig and replace the balls.
3. Start with the lowest circuit. Insert the balls into the nut circuit - the sleeve prevents the balls from falling out again.
4. Do not place the balls in the empty circuit located between the two deflectors.

If you have any technical queries please call **0333 207 4498.**

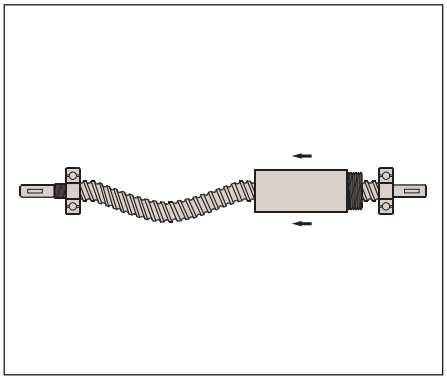
The maximum axial loads are the loads at which a ball screw will not buckle under compressive loads in the axial direction.

For ease of use the total maximum axial loads are shown in the following graphs.

Factors are:

- Type of supports
- Screw diameter (core)
- Length between supports

Apply 50% to the graph figures - to provide a margin of safety.



$$F_b = \frac{\alpha \bullet \eta_1 \bullet \pi^2 \bullet E \bullet I}{L^2}$$

Factors		
	η_1	η_2
Fixed-floating	2	10
Fixed-fixed	4	20
Fixed-free	0,25	1,3

F_b = buckling load (N)
α = safety factor 0.5
E = Young's modulus 2.06 x 10^5 (N/mm^2)
L = distance between supports
I = area moment of inertia

$$I = \frac{\pi}{64} \times d_c^4$$

d_c = Core ø of screw

A ball screw of size 40ø. This has a core diameter of 35.8mm.
Distance between the support parts is 600mm.
Using a fixed and floating bearing arrangement.
Reading of graph 1 (fixed-floating) the maximum axial load is going to be 50,000 N (approximately 5 tons).

Example:

Area moment of inertia $I = \frac{\pi}{64} \times d_c^4$

$= \frac{\pi}{64} \times 33.2^4$

$= 59.638 mm^4$

Buckling load calculation - Method 1

Safety factor
η_1 factor - see table above

$$F_b = \frac{0.5 \bullet 2 \bullet \pi^2 \bullet E \bullet I}{L^2}$$

$= 0.5 \times 2 \times 3.14^2 \times 600^2 (206{,}000) \times 59{,}638$
$= 337$ kN

Buckling load calculation - Method 2

$$F_b = \eta^2 \bullet \frac{d_c^4}{L^2} \bullet 10^4$$

η_2 factor - see table above

$$= 10 \times \frac{(33.2)^4 \times 10^4}{600^2}$$

$= 337$ kN

Buckling load - check graph
Gives max. load = 370kN